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IN THE SPECIFICATION:

Page 15, replace the paragraph starting at line 23 and ending at line 29 with the following paragraph.

The adjusting unit 4 is formed as a spindle drive in the present exemplified embodiment and has an actuating element to be driven to rotate, in the form of a stationary spindle 20 mounted to rotate, on which a spindle nut 22 is arranged torque proofly and moveable in the axial direction, which nut forms an adjusting element of adjusting unit 4 and loosely charges pivot lever 12 for pivoting the same. When the spindle 20 is driven to rotate, the spindle nut 22 moves, under the effect of the spindle thread, either to the right or the left in the drawing, according to the direction of rotation of spindle [[22]] 20.

Page 16, replace the paragraph starting at line 20 and ending at page 17, line 2 with the following paragraph.

In the exemplified embodiment represented in Fig. 1, the pinion side clutching element 30 is mounted to be moveable in the axial direction of spindles 20, 20', i.e. in the direction of the double arrow [[34]] 31 of Fig. 1, such that by axial movement of the pinion side clutching element 30 alternately an engagement of the power take-off side clutching element 32 of the adjusting unit 4 or the power take-off side clutching element 32' of adjusting unit 6, respectively, can be brought about with the pinion side clutching element 30. In order to clutch the pinion side clutching element 30 torque proofly to the respective power take-off side clutching element 32 and 32', respectively, in the respective coupling or clutching position, the pinion side clutching

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element 30 has, on its axial sides facing the power take-off side clutching elements, axial protrusions 34, 36, which engage recesses 38, 40 on the power take-off side clutching elements 32, 32', respectively, which recesses are formed complementary regarding the protrusions 34, 36. As it cannot be taken from the drawing and is thus explained herein, the axial protrusions arranged on one axial side of the pinion side clutching element 30 extend, in the peripheral direction of the clutching element 30, only over a short distance and are arranged at a peripheral distance from each other such that in this manner in the clutching position a positive engagement can be brought about between the pinion side clutching element 30 and the power take-off side clutching element 32 and 32', respectively.

Page 17, replace the paragraph starting at line 21 and ending at line 27 with the following paragraph.

In the first clutching or coupling position represented in Fig. 1, the pinion side or pinion end clutching element 30 is biased towards the left in Fig. 1 via the two-armed lever 42 and the tension spring 50, so that it engages the power take-off side clutching element of the adjusting unit 4. At this time, the protrusions 34 of the pinion end clutching element 30 engage the protrusions recesses 38 of the power take-off side clutching element 32 of adjusting unit 4, so that the pinion end clutching element 30 is torque proofly connected with the power take-off side clutching element 32 and thus with the spindle 20.

Page 23, replace the paragraph starting at line 23 and ending at line 34 with the following paragraph.

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In order to be able to make the power take-off side clutching element 32 of the adjusting unit 4 engage the pinion end clutching element 30, the pull-string 48 is pulled towards the left in Fig. 12, so that the lever 92 pivots clockwise and hereby moves the take-off side clutching element 32 in the axial direction towards the right in Fig. 12 and brings it into engagement with the pinion end clutching element 30 as has been shown in Fig. 13. As can be taken from Fig. 13, the axial lengths of the protrusions 96, 108 106, 108 are dimensioned such that in an axial position in which the take-off side clutching element 32 is already engaging the pinion end clutching element 30, the power take-off side clutching element 32 is still engaging the other clutching element 104, so that in this axial position represented in Fig. 13 the further clutching element 104 is connected in a torque proof manner with the pinion end clutching element 30. A resetting of the power take-off side clutching element 32 into the position represented in Fig. 12 occurs by the biasing force of tension spring 100.

Page 24, replace the paragraph starting at line 13 and ending at line 17 with the following paragraph.

In Fig. 15 a further clutching position has been represented, in which the take-off side clutching element 32 engages the further clutching element 104 and the take-off side clutching element [[32]] 30, while the take-off side clutching element 32' engages the pinion side clutching element 30 and the further clutching element 104', so that both adjusting units are clutched-in with the drive motor 24.